

J. Szantyr – Lecture No. 12 – Energy conservation equation

Kinetic energy of the fluid may be treated as the sum of the macroscopic motion energy and molecular motion energy or

internal energy: $\int_V \left(\frac{u^2}{2} + e \right) dV$ $\bar{u}(u_x, u_y, u_z)$ $u = |\bar{u}|$

Rate of change (i.e. material derivative) of the total kinetic energy of the fluid volume V surrounded by the fluid surface S is equal to the sum of the power of mass forces, the power of surface forces and the stream of energy (heat) supplied to the element.

$$\frac{D}{Dt} \int_V \rho \left(\frac{u^2}{2} + e \right) dV = \int_V \rho \bar{f} \cdot \bar{u} dV + \int_{s(V)} \bar{\tau} \cdot \bar{u} dS - \int_{s(V)} \bar{j} \cdot \bar{n} dS$$

where:

\bar{f}	unit mass force	$\bar{f}(f_x, f_y, f_z)$
$\bar{\tau}$	unit surface force	
\bar{j}	stream of supplied energy (heat)	$\bar{j}(j_x, j_y, j_z)$
\bar{n}	External unit normal vector	

The surface integrals in this equation may be converted into the volume integrals. If simultaneously we express the unit surface forces by the stress tensor $[\mathbf{P}]$, the above equation may be converted into the following form:

$$\int_{\bar{v}} \left[\rho \frac{D}{Dt} \left(\frac{u^2}{2} + e \right) - \rho \bar{f} \bullet \bar{u} - \text{div}([\mathbf{P}]\bar{u}) + \text{div}\bar{j} \right] dV = 0$$

As the volume of integration is arbitrarily selected, the function under the integral must be also equal to zero, what leads to the integral form of the energy conservation equation:

$$\rho \frac{D}{Dt} \left(\frac{u^2}{2} + e \right) = \rho \bar{f} \bullet \bar{u} + \text{div}([\mathbf{P}]\bar{u}) - \text{div}\bar{j}$$

Full development of the operators in the equation leads to the following form:

$$\rho \frac{D}{Dt} \left(\frac{u^2}{2} + e \right) = \rho (f_x u_x + f_y u_y + f_z u_z) + \frac{\partial}{\partial x} [(-p + \tau_{xx})u_x + \tau_{yx}u_y + \tau_{zx}u_z] +$$

$$+ \frac{\partial}{\partial y} [\tau_{xy}u_x + (-p + \tau_{yy})u_y + \tau_{zy}u_z] + \frac{\partial}{\partial z} [\tau_{xz}u_x + \tau_{yz}u_y + (-p + \tau_{zz})u_z] +$$

$$- \frac{\partial j_x}{\partial x} - \frac{\partial j_y}{\partial y} - \frac{\partial j_z}{\partial z}$$

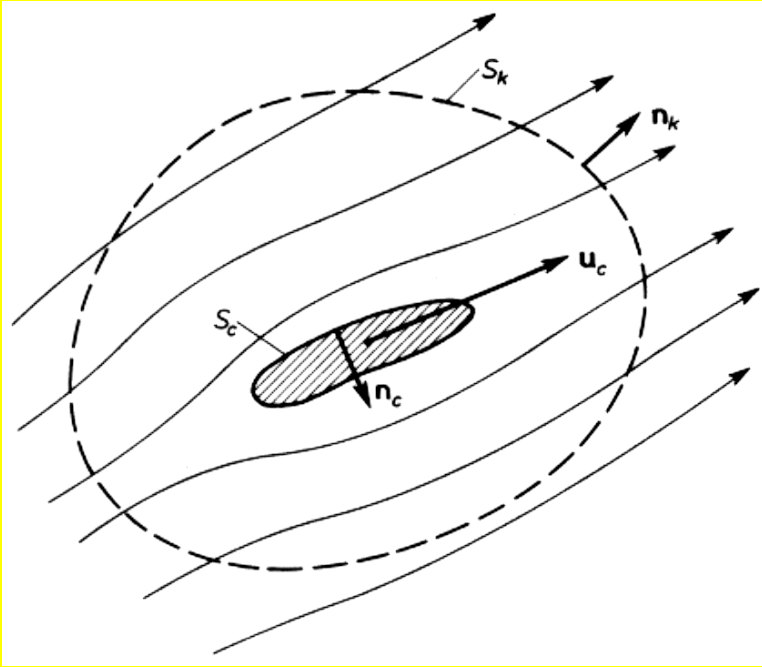
If the field of mass forces is a stationary potential field such that $\bar{f} = -grad\Pi$ then the energy conservation equation may be written as:

$$\rho \frac{D}{Dt} \left[\left(\frac{u^2}{2} + e + \Pi \right) \right] = div([P]\bar{u}) - div\bar{j}$$

or in the form of an integral equation:

$$\int_V \frac{\partial}{\partial t} \left[\rho \left(\frac{u^2}{2} + e + \Pi \right) \right] dV + \int_{s(V)} \rho \left(\frac{u^2}{2} + e + \Pi \right) u_n dS = \int_{s(V)} \bar{\tau} \bullet \bar{u} dS - \int_{s(V)} \bar{j} \bullet \bar{n} dS$$

Example No. 1: the case of an external flow around a body



S_k - the control surface

u_c - the velocity of the body

The energy conservation equation for such a case has the form:

$$\int_{V(S_k-S_c)} \frac{\partial}{\partial t} \left[\rho \left(\frac{u^2}{2} + e + \Pi \right) \right] dV + \int_{S_k} \rho \left(\frac{u^2}{2} + e + \Pi \right) u_n dS_k + \int_{S_c} \rho \left(\frac{u^2}{2} + e + \Pi \right) u_n dS_c =$$

$$= \int_{S_k} \bar{\tau}_{mn} \bullet \bar{u} dS_k + \int_{S_k} \bar{\tau}_{nt} \bullet \bar{u} dS_k + \int_{S_c} \bar{\tau}_{mn} \bullet \bar{u} dS_c + \int_{S_c} \bar{\tau}_{nt} \bullet \bar{u} dS_c - \int_{S_k} \bar{j} \bullet \bar{n}_k dS_k - \int_{S_c} \bar{j} \bullet \bar{n}_c dS_c$$

As the body surface is impermeable, the third term on the left hand side is equal zero. Moreover, if the velocity \bar{u}_c is the same in all points of the body, we may write:

$$\int_{S_c} \bar{\tau}_{nn} \bullet \bar{u}_c dS_c + \int_{S_c} \bar{\tau}_{nn} \bullet \bar{u}_c dS_c = \bar{u}_c \left[\int_{S_c} \bar{\tau}_{nn} dS_c + \int_{S_c} \bar{\tau}_{nt} dS_c \right] = -\bar{u}_c \bullet \bar{R}$$

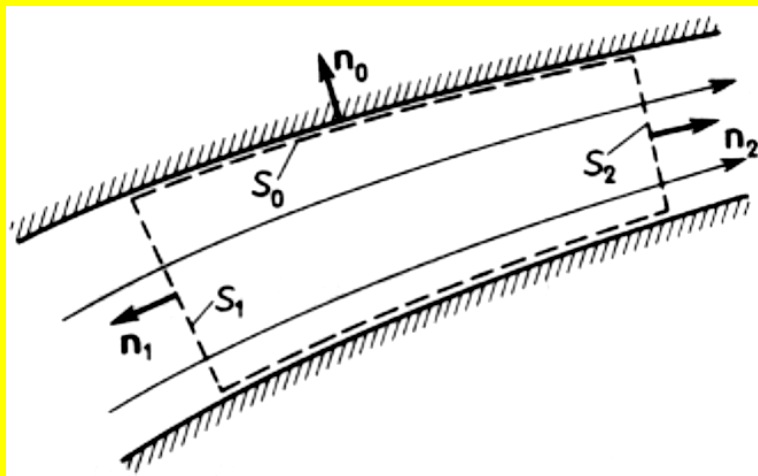
where R is the hydrodynamic reaction force acting on the body.

If we also assume: $\frac{\partial}{\partial t} = 0$ $\bar{j} = 0$ $\bar{\tau}_{nt} = 0$ $\bar{\tau}_{nn} = -p\bar{n}$

we obtain:
$$\bar{u}_c \bullet \bar{R} = - \int_{S_k} \rho \left(\frac{u^2}{2} + e + \frac{p}{\rho} + \Pi \right) u_n dS_k$$

Conclusion: the hydrodynamic force acting on a body may be determined through the analysis of the hydrodynamic wake behind this body, without the direct measurement e.g. by means of a dynamometer.

Example No. 2: the case of internal flow (flow in a channel)



Assumption: the flow is stationary.

The energy conservation equation for such a case has the form:

$$m \left[\left(\frac{\tilde{u}^2}{2} + \tilde{e} + \tilde{\Pi} \right)_{S_2} - \left(\frac{\tilde{u}^2}{2} + \tilde{e} + \tilde{\Pi} \right)_{S_1} \right] = \int_{S_0} \bar{\tau}_{mn} \cdot \bar{u} dS_0 + \int_{S_0} \bar{\tau}_{nt} \cdot \bar{u} dS_0 + \int_{S_1} \bar{\tau}_{mn} \cdot \bar{u} dS_1 +$$

$$+ \int_{S_1} \bar{\tau}_{nt} \cdot \bar{u} dS_1 + \int_{S_2} \bar{\tau}_{mn} \cdot \bar{u} dS_2 + \int_{S_2} \bar{\tau}_{nt} \cdot \bar{u} dS_2 - \int_{S_0} \bar{j} \cdot \bar{n}_0 dS_0 - \int_{S_1} \bar{j} \cdot \bar{n}_1 dS_1 - \int_{S_2} \bar{j} \cdot \bar{n}_2 dS_2$$

m – mass intensity of flow

The left hand side of the equation contains mean values defined in the following way:

$$\left(\frac{\tilde{u}^2}{2} + \tilde{e} + \tilde{\Pi} \right)_s = \frac{1}{m} \int_s \rho \left(\frac{u^2}{2} + e + \Pi \right) u_n dS$$

Additional assumptions:

-the fluid is inviscid, so the tangential stresses are zero and the normal stresses are expressed only by pressure,

-There is no heat transport through the inlet and outlet surfaces.

This leads to the expression:

where: $\Pi = gz$

$$m \left[\left(\frac{\tilde{u}^2}{2} + \tilde{e} + \frac{\tilde{p}}{\tilde{\rho}} + \tilde{\Pi} \right)_{s_1} - \left(\frac{\tilde{u}^2}{2} + \tilde{e} + \frac{\tilde{p}}{\tilde{\rho}} + \tilde{\Pi} \right)_{s_2} \right] = - \int_{s_0} \bar{j} \cdot \bar{n}_0 dS_0$$

This expression may describe for example the flow through the pipe in a heat exchanger.